

# EPRG programme—pipeline design for thick pipe



### EPRG Projects:

• 178/2015 Gap analysis for deep-water pipelines

• 202/2017 Thick-walled pipe under external overpressure

• 219/2019 Thick-walled pipe under internal overpressure

• 202a/2020 Improvement of collapse equation

- Paper about applicability of current design formulations for low values of D/t
  - Design governed by internal overpressure (burst) and bending
  - Design governed by external overpressure (collapse) and bending
- Cascading effect of improving collapse-pressure formulation on limit state for combined loading (LCC)





# Gap analysis and questionnaire results



Feedback on typical line-pipe specifications for 6-inch (SMLS) flowline, 16-inch (SMLS) gathering line and 24-inch (SAWL) trunkline, all requiring D/t < 15:

- Operators
  - Prospects requiring D/t < 15 for external-pressure design in water depths up to 3,700 m
- Pipe mills
  - D/t of 15 is generally at the limit of mills' capabilities for SAWL pipe
  - Manufacturing constraints relate to weldability of the seam weld and O-press capacity
  - For SMLS with small diameter, line-pipe specification necessary for ultra-deep-water application is typically within the pipe mills' existing capabilities
- Installation contractors
  - No direct limitation regarding the wall thickness
  - Girth welding using proven methods is feasible for wall thicknesses up to 50 mm

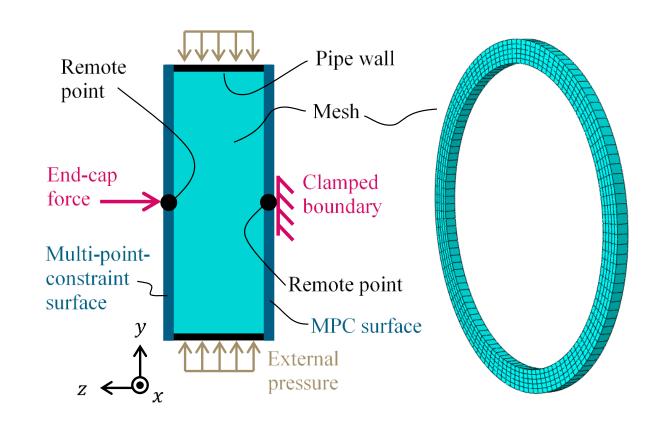




# Finite-element model used for benchmarking results



- Built using continuum elements (C3D20R) in Abaqus/Standard
- Continuous behaviour over length
- Non-linear material behaviour
- Geometric imperfections
  - Oval-shaped initial ovality
  - Eccentricity (for SMLS pipe)
- Bending applied by rotating ends





# Current limit state for combined loading



Limit state under load-controlled condition (LCC) in DNV-ST-F101 (2021)

External overpressure

$$\left[\gamma_{\text{m}} \gamma_{\text{SC,LB}} \frac{\left| M_{\text{Sd}} \right|}{\alpha_{\text{c}} M_{\text{p}}(t_{2})} + \left(\gamma_{\text{m}} \gamma_{\text{SC,LB}} \frac{S_{\text{Sd}}}{\alpha_{\text{c}} S_{\text{p}}(t_{2})}\right)^{2}\right]^{2} + \left[\gamma_{\text{m}} \gamma_{\text{SC,LB}} \frac{p_{\text{e}} - p_{\text{min}}}{p_{\text{c}}(t_{2})}\right]^{2} \leq 1$$

Internal overpressure

$$\left[\gamma_{\text{m}} \gamma_{\text{SC,LB}} \frac{\left| M_{\text{Sd}} \right|}{\alpha_{\text{c}} M_{\text{p}}(t_2)} + \left(\gamma_{\text{m}} \gamma_{\text{SC,LB}} \frac{S_{\text{Sd}}}{\alpha_{\text{c}} S_{\text{p}}(t_2)}\right)^2\right]^2 + \left[\gamma_{\text{p}} \frac{p_{\text{i}} - p_{\text{e}}}{\alpha_{\text{c}} p_{\text{b}}(t_2)}\right]^2 \le 1$$

Applicability range in previous design-code revisions bounded to  $15 \le D/t \le 45$ .

Lower-bound limit has been removed by DNV based on EPRG's work



### Definitions of cases



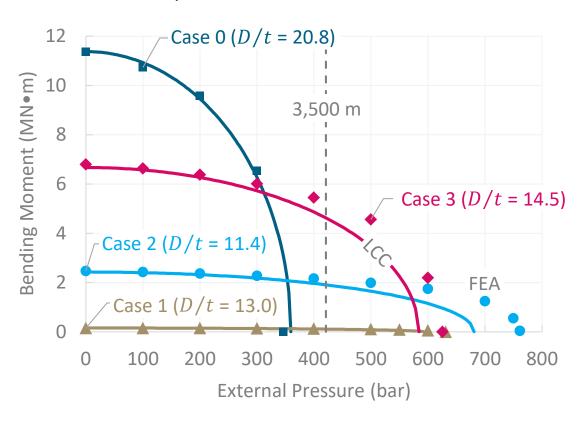
	Case 0	Case 1	Case 2	Case 3
Туре	Export	Flowline	Gathering	Trunkline
Material grade	DNV GL SAWL 450	DNV GL SMLS 450	DNV GL SMLS 450	DNV GL SAWL 450
Outer diameter	32 inch (812.8 mm)	6 5/8 inch (168.3 mm)	16 inch (406.4 mm)	24 inch (609.6 mm)
Wall thickness - External-pressure design - Internal-pressure design	39.0 mm	12.9 mm 21.1 mm	35.6 mm 36.6 mm	42.0 mm
<ul><li>D/t</li><li>External-pressure design</li><li>Internal-pressure design</li></ul>	20.8	13.0 8.0	11.4 11.1	14.5

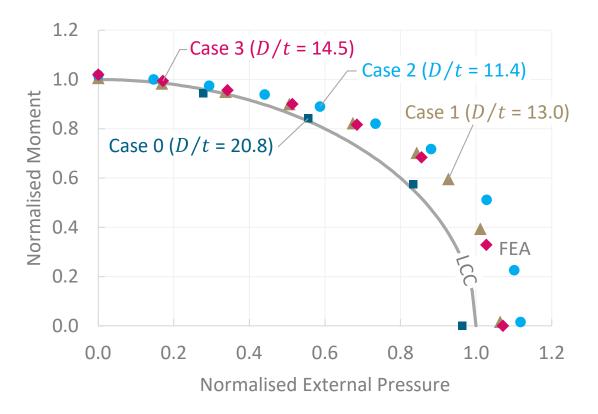


# Accuracy of current design formulations



### External overpressure



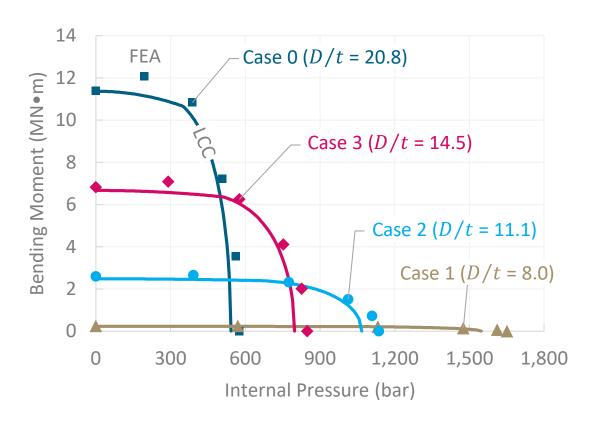


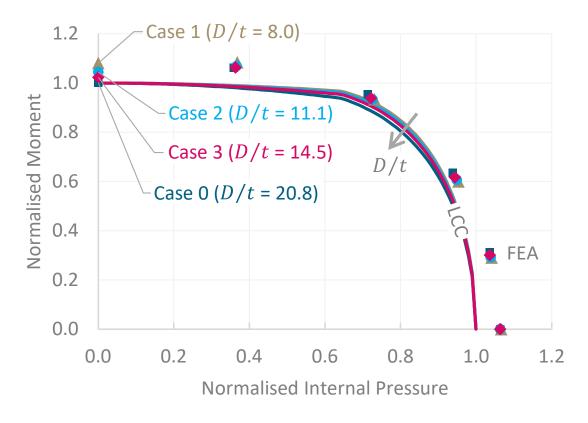


# Accuracy of current design formulations



### Internal overpressure





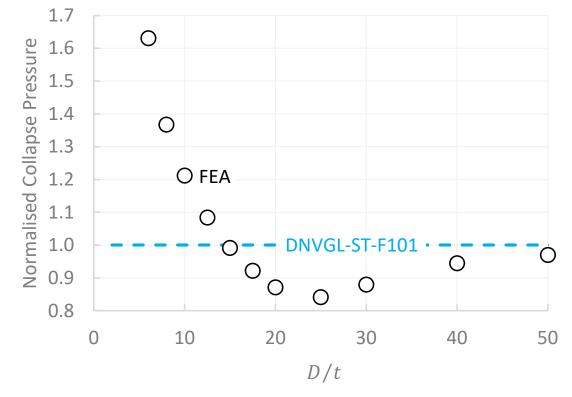


# Accuracy of current design formulations



Design equation for collapse compared with FEA results (for cold-formed pipe with no recovery of Bauschinger effect)

- $D/t \leq 15$ 
  - (Overly) conservative
- $15 < D/t \le 30-40$ 
  - Nonconservative
- D/t > 30-40
  - Good agreement





# Improved formulation for collapse



Transcendental equation to be solved for collapse pressure  $p_{
m C}$ 

$$[p_{bif}(p_{c}) - p_{c}](p_{p}^{2} - p_{c}^{2}) = p_{c} p_{bif}(p_{c}) p_{p} \frac{D}{t} O_{0} \psi \sqrt{1 - \frac{1}{4} \left(\frac{p_{c}}{p_{p}}\right)^{2}}$$

Triaxiality factor

$$\psi = \frac{2}{\sqrt{3}} \frac{D - t}{D - 2t}$$

Plastic collapse pressure

$$p_{\rm p} = \frac{2 t \sigma_{\rm y}}{D} \psi$$

Bifurcation pressure

$$p_{\text{bif}}(p_{\text{e}}) = \frac{{}^{2E}t(\sigma_{\text{eqv}}(p_{\text{e}}))}{1-\nu^2} \left(\frac{t}{D}\right)^3$$

Equivalent stress

$$\sigma_{\text{eqv}}(p_{\text{e}}) = \frac{p_{\text{e}}D}{2t} \frac{1}{\psi}$$

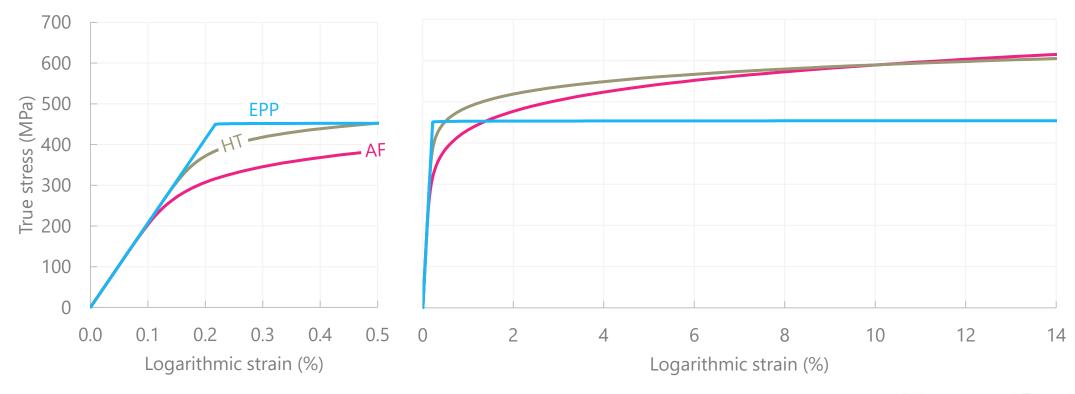
# Stress-strain curves (DNV 450)



#### Material conditions

- As-fabricated:
- Heat-treated:
- Elastic-perfectly plastic:

cold-formed pipe, yield stress reduced by 15% ( $\alpha_{fab}$  = 0.85) cold-formed pipe, after coating or (light) heat treatment typical for virgin steel, representative of seamless pipe

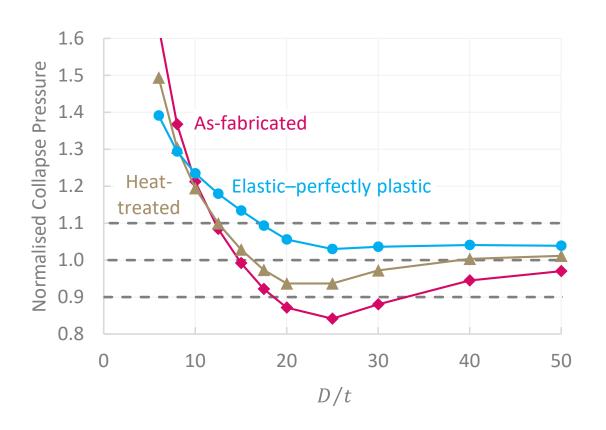


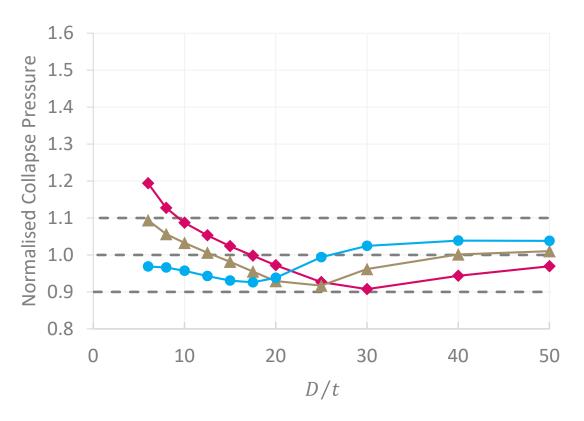




# Accuracy of improved collapse-pressure formulation

Proposed (right) analytical prediction much closer to finite-element results than current (left)



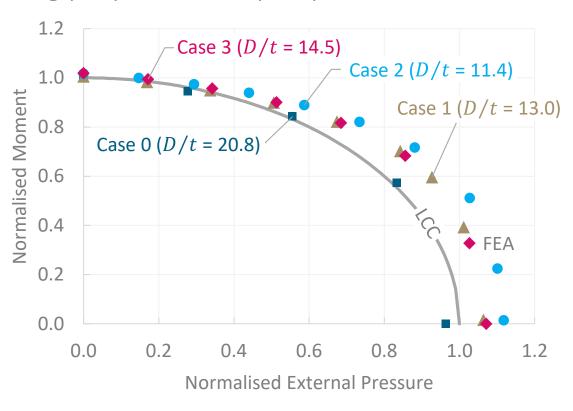


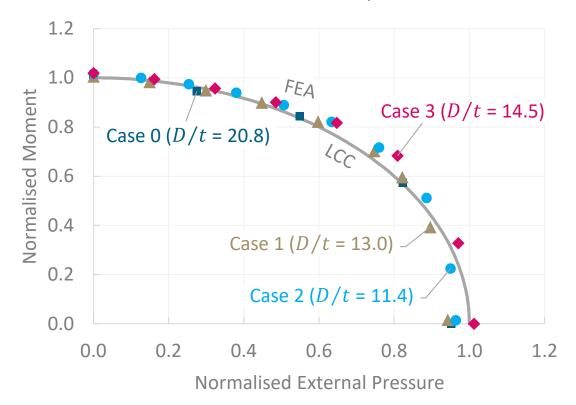


### Current v proposed



### Using proposed collapse-pressure formulation in LCC leads to much better prediction







### Conclusions



- Current collapse-pressure formulation conservative for D/t < 15
  - Lower-bound applicability limit of LCC criterion for external overpressure (D/t > 15) is unnecessary
- Predictive accuracy LCC criterion (internal overpressure) independent of D/t
  - Lower-bound applicability limit of LCC criterion for internal overpressure (D/t > 15) is unnecessary too
- DNV has removed this limit in DNV-ST-F101 (2021) and acknowledged EPRG's work
- To reduce conservatism in external-pressure design, framework for predicting collapse pressure of thick-walled pipe has been improved
  - Based on fundamental mechanisms underlying collapse (geometrical instability and material yielding)
- Proposed improvement of collapse-pressure formulation leads to cascading improvement of current LCC equation for external overpressure

### Recommendations



- FEA results have been used to benchmark analytical results. Recommended to confirm results using medium- or full-scale tests
  - Especially for very low D/t, for which not much testing has been done
- Influence of effective axial force has been ignored. Recommended to investigate this effect
- When adopting new design formulations, associated safety factors should be recalibrated



Thank you for your attention.